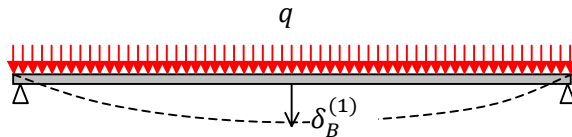
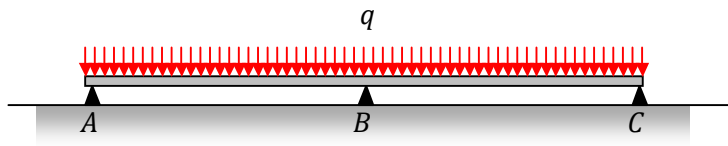
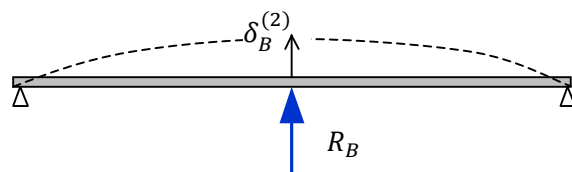


Problema 1.

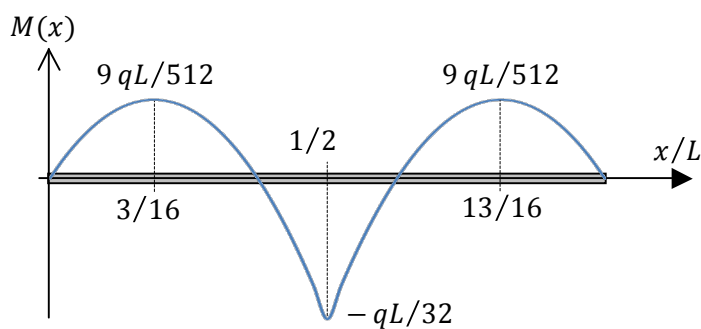
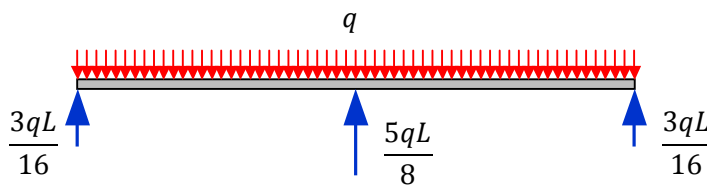


$$\delta_B^{(1)} = \frac{5qL^4}{384EI}$$



$$\delta_B^{(2)} = \frac{R_B L^4}{48EI}$$

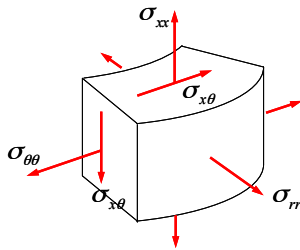
$$\delta_B^{(1)} = \delta_B^{(2)} \Rightarrow \frac{R_B L^3}{48EI} = \frac{5qL^4}{384EI} \Rightarrow R_B = \frac{5qL}{8}$$



$$M(x) = \begin{cases} \frac{3qL}{16} \left[\left(\frac{x}{L} \right) - \frac{8}{3} \left(\frac{x}{L} \right)^2 \right] & x \leq L/2 \\ \frac{3qL}{16} \left[\left(\frac{L-x}{L} \right) - \frac{8}{3} \left(\frac{L-x}{L} \right)^2 \right] & x > L/2 \end{cases}$$

$$\max\{|\sigma_{xx}(x, y)|\} = \frac{h}{2} \frac{\max\{|M(x)|\}}{bh^3/12} = \frac{6}{bh^2} \frac{qL}{32} = \frac{3}{16} \frac{qL}{bh^2} < S_Y \Rightarrow q < \frac{16}{3} \left(\frac{S_Y bh^2}{L} \right)$$

Problema 2.



$\sigma_{\theta\theta}$ - Tensão circunferencial devido à pressão interna

σ_{rr} - Tensão radial devido à pressão interna

σ_{xx} - Tensão axial devido à pressão interna

$\sigma_{x\theta}$ - Tensão cisalhante devido à torção

$$[\sigma] = \begin{bmatrix} \sigma_{xx} & \sigma_{x\theta} & 0 \\ \sigma_{x\theta} & \sigma_{\theta\theta} & 0 \\ 0 & 0 & \sigma_{rr} \end{bmatrix}$$

Tensão	Parede Interna do Tubo	Parede Externa do Tubo
$\sigma_{\theta\theta}$	$\sigma_{\theta\theta} = \frac{(b^2/a^2 + 1)}{(b^2/a^2 - 1)} P = 128 \text{ MPa}$	$\sigma_{\theta\theta} = \frac{2}{(b^2/a^2 - 1)} P = 101 \text{ MPa}$
σ_{rr}	$\sigma_{rr} = -P = -27.6 \text{ MPa}$	$\sigma_{rr} = 0$
σ_{xx}	$\sigma_{xx} = \frac{1}{(b^2/a^2 - 1)} P = 50.4 \text{ MPa}$	$\sigma_{xx} = \frac{1}{(b^2/a^2 - 1)} P = 50.4 \text{ MPa}$
$\sigma_{x\theta}$	$\sigma_{x\theta} = a \frac{T}{\pi(b^4 - a^4)/2} = 22.8 \text{ MPa}$	$\sigma_{x\theta} = b \frac{T}{\pi(b^4 - a^4)/2} = 28.4 \text{ MPa}$
σ_M	$\sigma_M = \frac{\sigma_{\theta\theta} + \sigma_{xx}}{2} = 89.4 \text{ MPa}$	$\sigma_M = \frac{\sigma_{\theta\theta} + \sigma_{xx}}{2} = 75.6 \text{ MPa}$
R	$R = \sqrt{\left(\frac{\sigma_{\theta\theta} - \sigma_{xx}}{2}\right)^2 + \sigma_{x\theta}^2} = 45.2 \text{ MPa}$	$R = \sqrt{\left(\frac{\sigma_{\theta\theta} - \sigma_{xx}}{2}\right)^2 + \sigma_{x\theta}^2} = 38.0 \text{ MPa}$
σ_1	$\sigma_1 = \sigma_M + R = 135 \text{ MPa}$	$\sigma_1 = \sigma_M + R = 114 \text{ MPa}$
σ_2	$\sigma_2 = \sigma_M - R = 44.2 \text{ MPa}$	$\sigma_2 = \sigma_m - R = 37.7 \text{ MPa}$
σ_3	$\sigma_3 = \sigma_{rr} = -27.6 \text{ MPa}$	$\sigma_3 = \sigma_{rr} = 0.0$
σ_{VM}	$\sigma_{VM} = 141 \text{ MPa}$	$\sigma_{VM} = 100 \text{ MPa}$
$\tau_{\max}$	$\tau_{\max} = 81.1 \text{ MPa}$	$\tau_{\max} = 56.8 \text{ MPa}$

A parede interna é a mais carregada. Portanto, os fatores de segurança são:

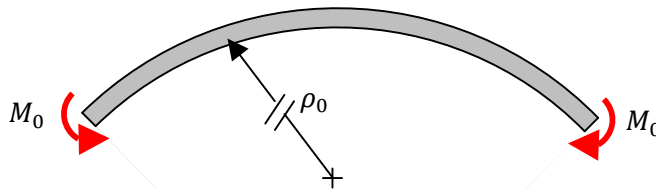
$$FS_{VM} = \frac{S_Y}{\sigma_{VM}} = 2.06$$

$$FS_{Tresca} = \frac{S_Y/2}{\tau_{\max}} = 1.79$$

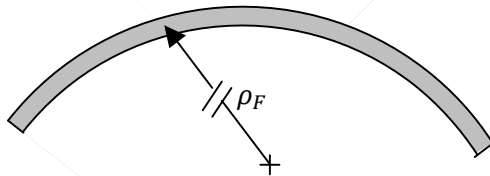
Problema 3.



$$M(\kappa) = \begin{cases} \kappa EI, & \kappa \leq \kappa_Y \\ \alpha M_Y \left[1 - \left(\frac{\alpha - 1}{\alpha} \right) \left(\frac{\kappa_Y}{\kappa} \right)^2 \right], & \kappa > \kappa_Y \end{cases}$$



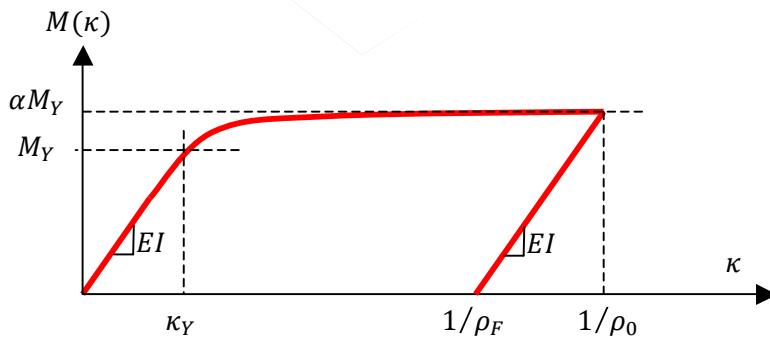
$$\sigma_{xx}(y) = \begin{cases} -S_Y, & -c \leq y < -r_Y \\ y S_Y / r_Y, & -r_Y \leq y < r_Y \\ S_Y, & r_Y \leq y < c \end{cases}$$



$$M_Y = S_Y I / c$$

$$\kappa_Y = M_Y / EI = S_Y / Ec$$

$$r_Y = c(\kappa_Y / \kappa)$$



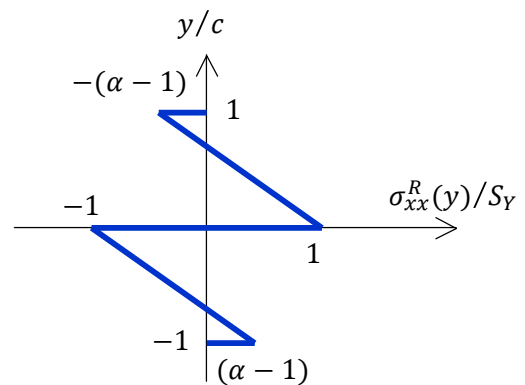
$$\kappa = 1/\rho$$

$$\frac{1}{\rho_0} - \frac{1}{\rho_F} \cong \frac{\alpha M_Y}{EI} \quad (1/\rho_F \gg \kappa_Y)$$

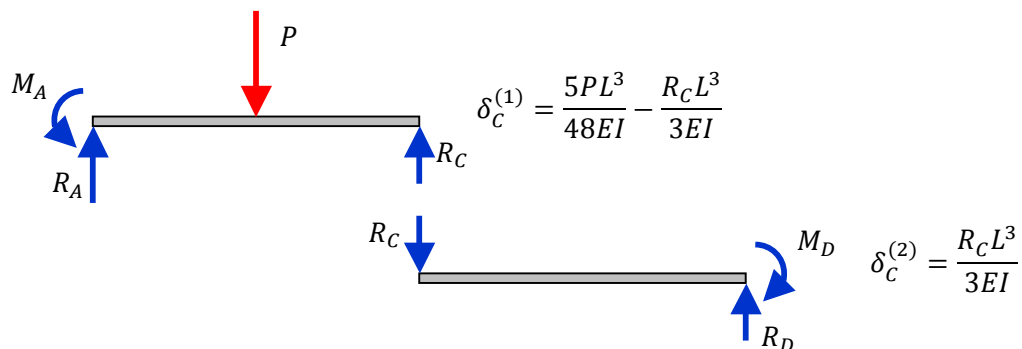
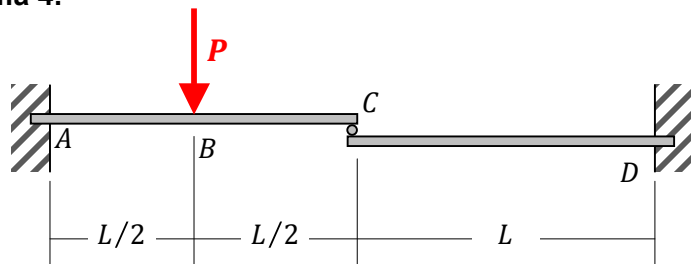
$$\rho_0 = \left(\frac{1}{\rho_F} + \frac{\alpha M_Y}{EI} \right)^{-1} = 76.4 \text{ mm}$$

Distribuição da tensão residual:

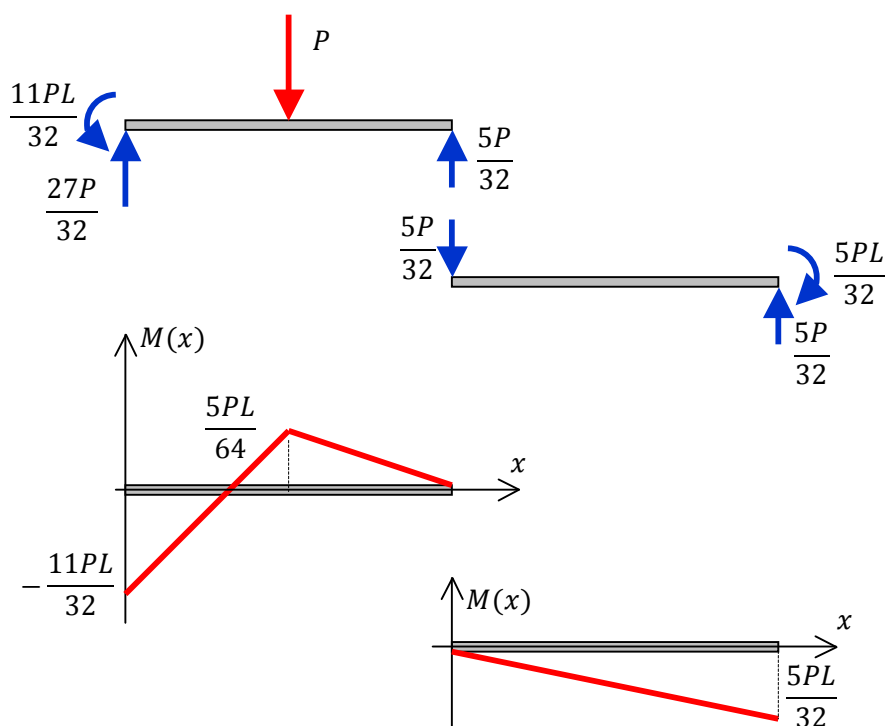
$$\sigma_{xx}^R(y) = \begin{cases} -S_Y - y \alpha M_Y / I, & -c \leq y < 0 \\ S_Y - y \alpha M_Y / I, & 0 \leq y < c \end{cases}$$



Problema 4.



$$\delta_C^{(1)} = \delta_C^{(2)} \Rightarrow \frac{5PL^3}{48EI} - \frac{R_C L^3}{3EI} = \frac{R_C L^3}{3EI} \Rightarrow R_C = \frac{5P}{32}$$



Escoamento na viga ABC

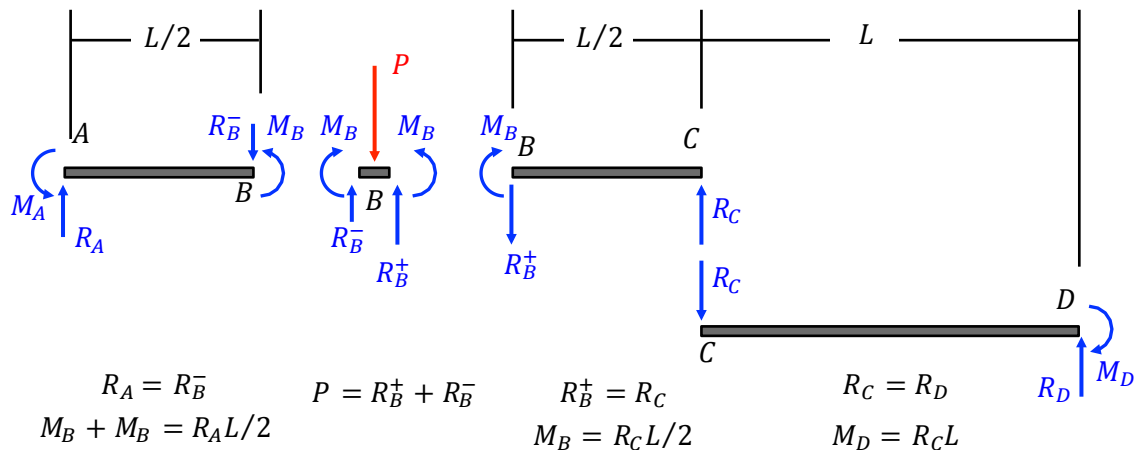
$$\max\{|\sigma_{xx}(x,y)|\} = \frac{h}{2} \frac{\max\{|M(x)|\}}{bh^3/12} = \frac{6}{bh^2} \frac{11PL}{32} = \frac{33PL}{16bh^2} < S_Y \Rightarrow P_Y = \frac{16S_Y bh^2}{33L}$$

Há duas possibilidades de colapso plástico:

(a) Rótulas plásticas em A e B

(b) Rótulas plásticas em A e D

Equilíbrio:



Hipótese (a), rótulas plásticas em A e B: $M_A = M_B = M_L$

$$R_C = 2M_B/L = 2M_L/L$$

$$R_A = P_L - R_C = P_L - 2M_B/L = P_L - 2M_L/L$$

$$M_A + M_B = 2M_L \Rightarrow 2M_L = (P_L - 2M_L/L)L/2 = P_L L/2 - M_L$$

$$\text{logo } P_L = 6M_L/L$$

Hipótese (b), rótulas plásticas em A e D: $M_A = M_D = M_L$

$$R_C = M_D/L = M_L/L$$

$$M_B = R_C L/2 = M_L/2$$

$$R_A = P_L - R_C = P_L - M_L/L$$

$$M_A + M_B = 3M_L/2 \Rightarrow 3M_L/2 = (P_L - M_L/L)L/2 = P_L L/2 - M_L/2$$

$$\text{logo } P_L = 4M_L/L$$

A carga de colapso plástico será a menor entre as calculadas para as hipóteses (a) e (b).

Portanto:

$$P_L = 4 \frac{M_L}{L}$$